

Figure 16.13 Geometry factors J for straight bevel gears. Pressure angle = 20° , shaft angle = 90° . (From AGMA Information Sheet 226.01; also see ANSI/AGMA 2003-A86.)

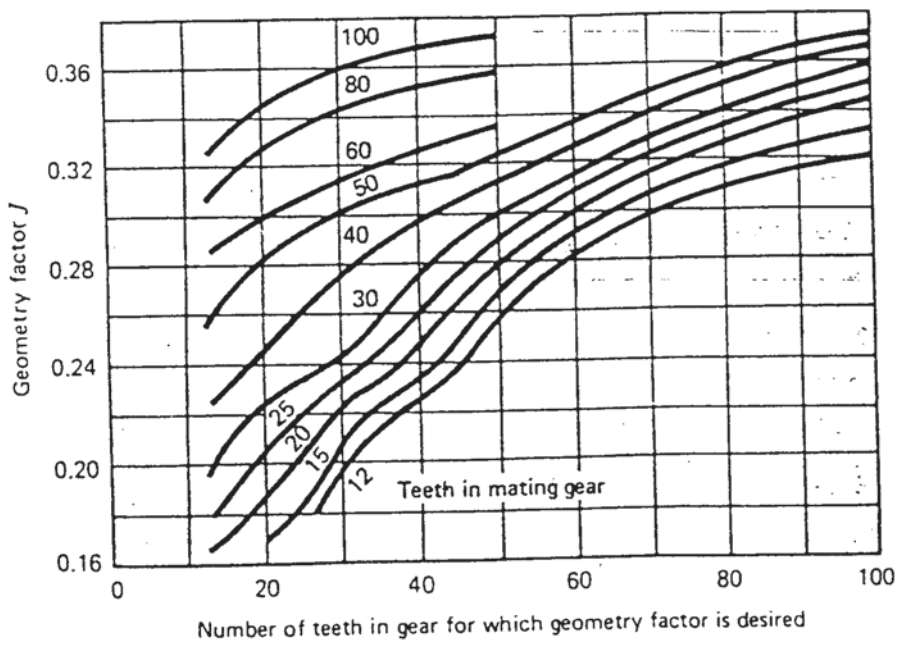


Figure 16.14 Geometry factors J for spiral bevel gears. Pressure angle = 20° , spiral angle = 35° , shaft angle = 90° . (From AGMA Information Sheet 226.01; also see ANSI/AGMA 2003-A86.)

(d)

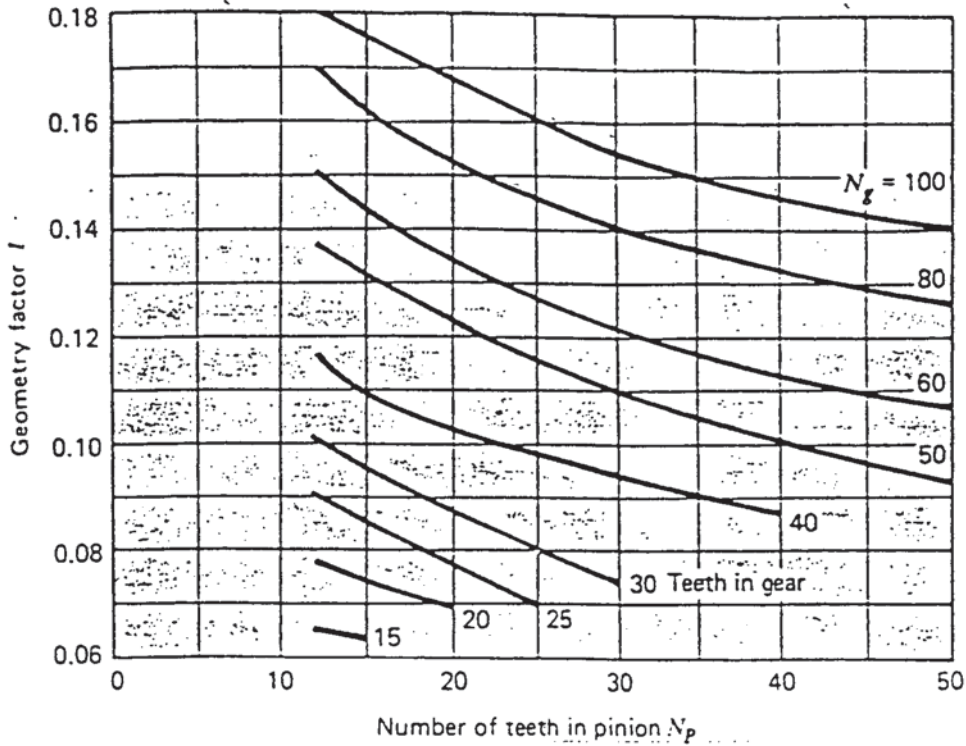


Figure 16.16 Geometry factors I for spiral bevel gears. Pressure angle = 20° , spiral angle = 35° , shaft angle = 90° . (From AGMA Information Sheet 215.91; also see ANSI/AGMA 2003-A86.)

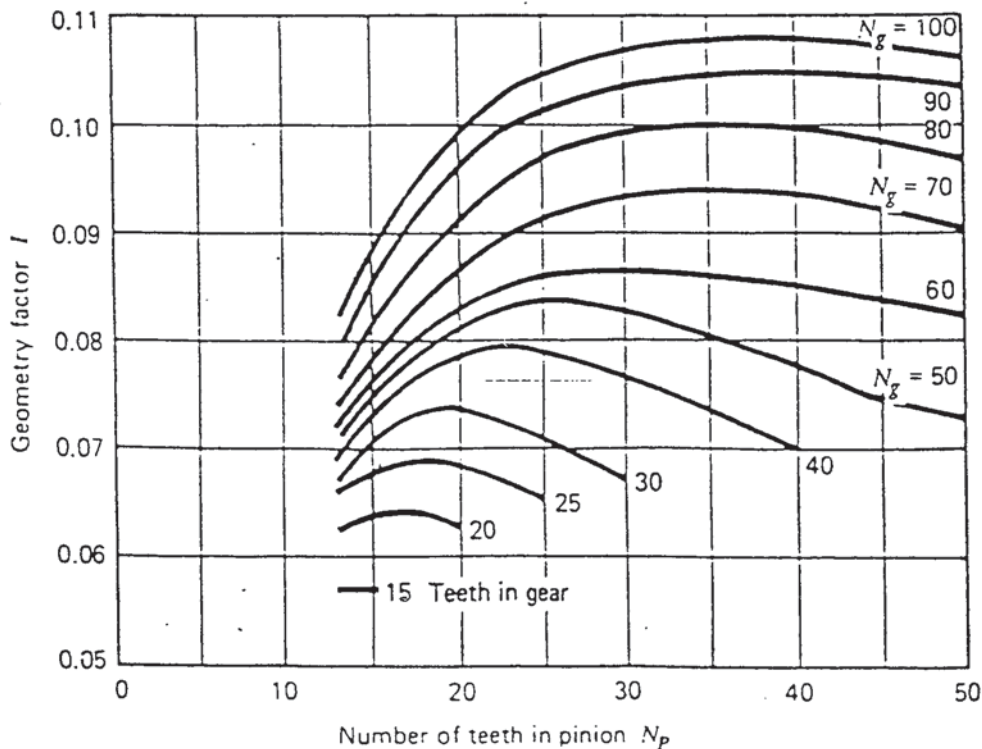


Figure 16.15 Geometry factors I for straight bevel gears. Pressure angle = 20° , shaft angle = 90° . (From AGMA Information Sheet 215.91; also see ANSI/AGMA 2003-A86.)

TABLE 17-6
Standard V-Belt Sections



| BELT SECTION | WIDTH a, in | THICKNESS b, in | MINIMUM SHEAVE DIAMETER, in | hp RANGE, ONE OR MORE BELTS |
|--------------|-------------|-----------------|-----------------------------|-----------------------------|
| A | 1/2 | 11/16 | 3.0 | 1-10 |
| B | 3/4 | 7/16 | 5.4 | 1-25 |
| C | 1 | 1/2 | 9.0 | 15-100 |
| D | 1 1/4 | 5/8 | 13.0 | 50-250 |
| E | 1 1/2 | 1 | 21.6 | 100 and up |

TABLE 17-7
Inside Circumferences of Standard V Belts

| SECTION | CIRCUMFERENCE, in |
|---------|---|
| A | 26, 31, 33, 35, 38, 42, 46, 48, 51, 53, 55, 57, 60, 62, 64, 66, 68, 71, 75, 78, 80, 85, 90, 96, 105, 112, 120, 128 |
| B | 35, 38, 42, 46, 48, 51, 53, 55, 57, 60, 62, 64, 65, 66, 68, 71, 75, 78, 79, 81, 83, 85, 90, 93, 97, 100, 103, 105, 112, 120, 128, 131, 136, 144, 158, 173, 180, 195, 210, 240, 270, 300 |
| C | 51, 60, 68, 75, 81, 85, 90, 96, 105, 112, 120, 128, 136, 144, 158, 162, 173, 180, 195, 210, 240, 270, 300, 330, 360, 390, 420 |
| D | 120, 128, 144, 158, 162, 173, 180, 195, 210, 240, 270, 300, 330, 360, 390, 420, 480, 540, 600, 660 |
| E | 180, 195, 210, 240, 270, 300, 330, 360, 390, 420, 480, 540, 600, 660 |

FIGURE 17-7
Correction factor K_1 for angle of contact. Multiply the rated horsepower per belt by this factor to obtain the corrected horsepower.

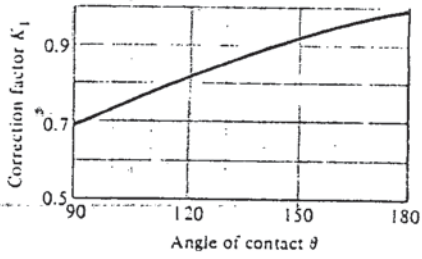


TABLE 17-10
Belt-Length Correction Factor K_2 *

| LENGTH FACTOR | NOMINAL BELT LENGTH, in | | | | |
|---------------|-------------------------|------------|------------|------------|-----------|
| | A BELTS | B BELTS | C BELTS | D BELTS | E BELTS |
| 0.85 | Up to 35 | Up to 46 | Up to 75 | Up to 128 | |
| 0.90 | 38-46 | 48-60 | 81-96 | 144-162 | Up to 195 |
| 0.95 | 48-55 | 62-75 | 105-120 | 173-210 | 210-240 |
| 1.00 | 60-75 | 78-97 | 128-158 | 240 | 270-300 |
| 1.05 | 78-90 | 105-120 | 162-195 | 270-330 | 330-390 |
| 1.10 | 96-112 | 128-144 | 210-240 | 360-420 | 420-480 |
| 1.15 | 120 and up | 158-180 | 270-300 | 480 | 540-600 |
| 1.20 | | 195 and up | 330 and up | 540 and up | 660 |

*Multiply the rated horsepower per belt by this factor to obtain the corrected horsepower.

TABLE 17-11
Suggested Service Factors K_S for V-Belt Drives

| DRIVEN MACHINERY | SOURCE OF POWER | |
|------------------|------------------------------|---------------------------|
| | NORMAL TORQUE CHARACTERISTIC | HIGH OR NONUNIFORM TORQUE |
| Uniform | 1.0 to 1.2 | 1.1 to 1.3 |
| Light shock | 1.1 to 1.3 | 1.2 to 1.4 |
| Medium shock | 1.2 to 1.4 | 1.4 to 1.6 |
| Heavy shock | 1.3 to 1.5 | 1.5 to 1.8 |

(V)

TABLE 17-9

Horsepower Ratings of
Standard V Belts

| BELT SECTION | SHEAVE PITCH DIAMETER, in | BELT SPEED, ft/min | | | | |
|--------------|---------------------------|--------------------|------|------|------|------|
| | | 1000 | 2000 | 3000 | 4000 | 5000 |
| A | 2.6 | 0.47 | 0.62 | 0.53 | 0.15 | |
| | 3.0 | 0.66 | 1.01 | 1.12 | 0.93 | 0.38 |
| | 3.4 | 0.81 | 1.31 | 1.57 | 1.53 | 1.12 |
| | 3.8 | 0.93 | 1.55 | 1.92 | 2.00 | 1.71 |
| | 4.2 | 1.03 | 1.74 | 2.20 | 2.38 | 2.19 |
| | 4.6 | 1.11 | 1.89 | 2.44 | 2.69 | 2.58 |
| | 5.0 and up | 1.17 | 2.03 | 2.64 | 2.96 | 2.89 |
| B | 4.2 | 1.07 | 1.58 | 1.68 | 1.26 | 0.22 |
| | 4.6 | 1.27 | 1.99 | 2.29 | 2.08 | 1.24 |
| | 5.0 | 1.44 | 2.33 | 2.80 | 2.76 | 2.10 |
| | 5.4 | 1.59 | 2.62 | 3.24 | 3.34 | 2.82 |
| | 5.8 | 1.72 | 2.87 | 3.61 | 3.85 | 3.45 |
| | 6.2 | 1.82 | 3.09 | 3.94 | 4.28 | 4.00 |
| | 6.6 | 1.92 | 3.29 | 4.23 | 4.67 | 4.48 |
| | 7.0 and up | 2.01 | 3.46 | 4.49 | 5.01 | 4.90 |
| C | 6.0 | 1.84 | 2.66 | 2.72 | 1.87 | |
| | 7.0 | 2.48 | 3.94 | 4.64 | 4.44 | 3.12 |
| | 8.0 | 2.96 | 4.90 | 6.09 | 6.36 | 5.52 |
| | 9.0 | 3.34 | 5.65 | 7.21 | 7.86 | 7.39 |
| | 10.0 | 3.64 | 6.25 | 8.11 | 9.06 | 8.89 |
| | 11.0 | 3.88 | 6.74 | 8.84 | 10.0 | 10.1 |
| | 12.0 and up | 4.09 | 7.15 | 9.46 | 10.9 | 11.1 |
| D | 10.0 | 4.14 | 6.13 | 6.55 | 5.09 | 1.35 |
| | 11.0 | 5.00 | 7.83 | 9.11 | 8.50 | 5.62 |
| | 12.0 | 5.71 | 9.26 | 11.2 | 11.4 | 9.18 |
| | 13.0 | 6.31 | 10.5 | 13.0 | 13.8 | 12.2 |
| | 14.0 | 6.82 | 11.5 | 14.6 | 15.8 | 14.8 |
| | 15.0 | 7.27 | 12.4 | 15.9 | 17.6 | 17.0 |
| | 16.0 | 7.66 | 13.2 | 17.1 | 19.2 | 19.0 |
| | 17.0 and up | 8.01 | 13.9 | 18.1 | 20.6 | 20.7 |
| E | 16.0 | 8.68 | 14.0 | 17.5 | 18.1 | 15.3 |
| | 18.0 | 9.92 | 16.7 | 21.2 | 23.0 | 21.5 |
| | 20.0 | 10.9 | 18.7 | 24.2 | 26.9 | 26.4 |
| | 22.0 | 11.7 | 20.3 | 26.6 | 30.2 | 30.5 |
| | 24.0 | 12.4 | 21.6 | 28.6 | 32.9 | 33.8 |
| | 26.0 | 13.0 | 22.8 | 30.3 | 35.1 | 36.7 |
| | 28.0 and up | 13.4 | 23.7 | 31.8 | 37.1 | 39.1 |

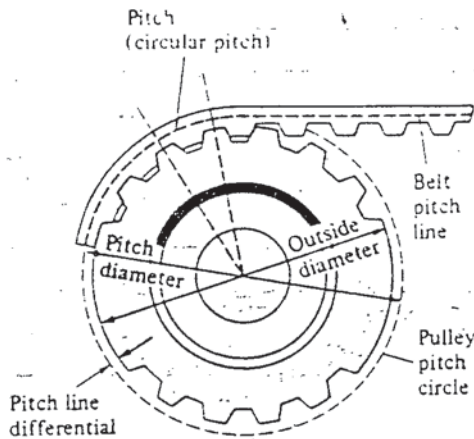


FIGURE 21.8
The concept of pitch diameter of a positive drive sprocket. (Illustration courtesy Goodyear Industrial Products Division etc.)

TABLE 21.4
Positive drive belts

| Pitch, in | Working tension, lb/in width | Weight, lb/ft (1 in wide) |
|----------------|------------------------------|---------------------------|
| 0.08 | 32 | 0.016 |
| 0.20 | 41 | 0.046 |
| $\frac{1}{8}$ | 55 | 0.064 |
| $\frac{1}{4}$ | 140 | 0.090 |
| $\frac{3}{8}$ | 191 | 0.210 |
| $1\frac{1}{2}$ | 234 | 0.27 |

Courtesy Goodyear Tire and Rubber Company

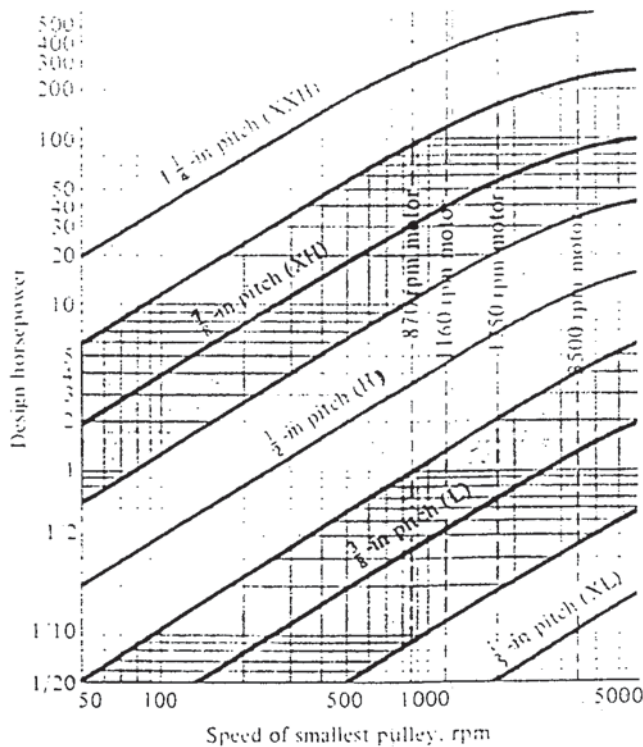


FIGURE 21.9
Typical guide for selection of positive drive belts. (Courtesy Goodyear Industrial Products.)

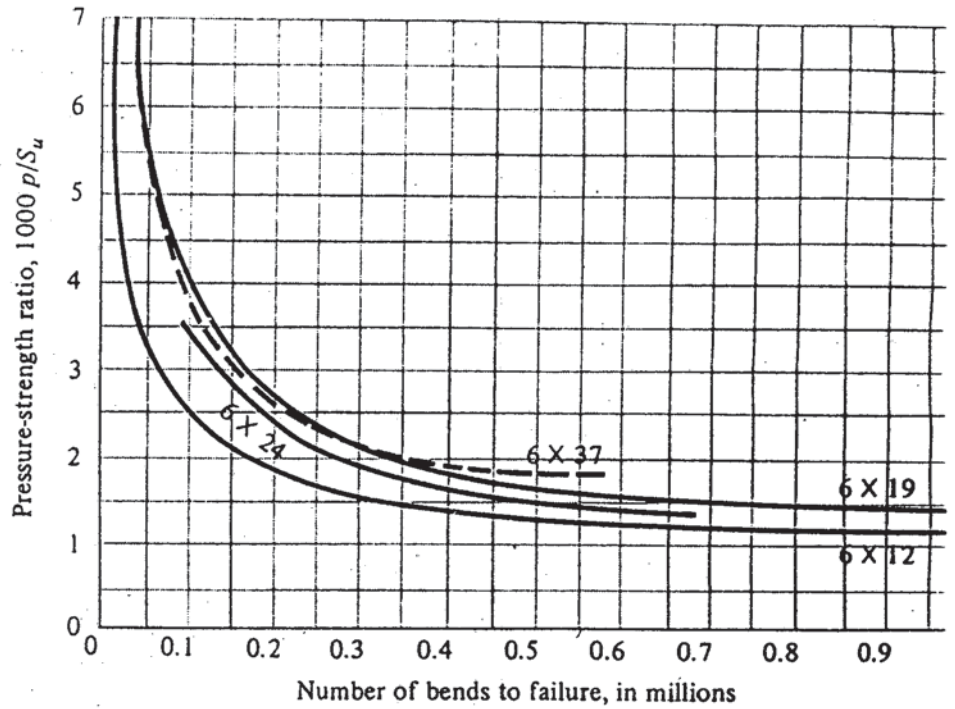


FIGURE 17-14

Experimentally determined relation between the fatigue life of wire rope and the sheave pressure.

TABLE 17-18
Wire-Rope Data

| ROPE | WEIGHT PER FOOT, lb | MINIMUM SHEAVE DIAMETER, in | STANDARD SIZES d , in | MATERIAL | SIZE OF OUTER WIRES | MODULUS OF ELASTICITY,* Mpsi | STRENGTH,† kpsi |
|--------------------------|---------------------|-----------------------------|---------------------------------|---------------------------|---------------------|------------------------------|-----------------|
| 6 × 7 haulage | $1.50d^2$ | $.42d$ | $\frac{1}{4}$ – $1\frac{1}{2}$ | Monitor steel | $d/9$ | 14 | 100 |
| | | | | Plow steel | $d/9$ | 14 | 88 |
| | | | | Mild plow steel | $d/9$ | 14 | 76 |
| 6 × 19 standard hoisting | $1.60d^2$ | $26d$ – $34d$ | $\frac{1}{4}$ – $2\frac{3}{4}$ | Monitor steel | $d/13$ – $d/16$ | 12 | 106 |
| | | | | Plow steel | $d/13$ – $d/16$ | 12 | 93 |
| | | | | Mild plow steel | $d/13$ – $d/16$ | 12 | 80 |
| 6 × 37 special flexible | $1.55d^2$ | $18d$ | $\frac{1}{4}$ – $3\frac{1}{2}$ | Monitor steel | $d/22$ | 11 | 100 |
| | | | | Plow steel | $d/22$ | 11 | 88 |
| 8 × 19 extra flexible | $1.45d^2$ | $21d$ – $26d$ | $\frac{1}{4}$ – $1\frac{1}{2}$ | Monitor steel | $d/15$ – $d/19$ | 10 | 92 |
| | | | | Plow steel | $d/15$ – $d/19$ | 10 | 80 |
| 7 × 7 aircraft | $1.70d^2$ | — | $\frac{1}{8}$ – $\frac{3}{8}$ | Corrosion-resistant steel | — | — | 124 |
| | | | | Carbon steel | — | — | 124 |
| 7 × 9 aircraft | $1.75d^2$ | — | $\frac{1}{8}$ – $1\frac{1}{8}$ | Corrosion-resistant steel | — | — | 135 |
| | | | | Carbon steel | — | — | 143 |
| 19-wire aircraft | $2.15d^2$ | — | $\frac{3}{32}$ – $\frac{5}{16}$ | Corrosion-resistant steel | — | — | 165 |
| | | | | Carbon steel | — | — | 165 |

*The modulus of elasticity is only approximate; it is affected by the loads on the rope and, in general, increases with the life of the rope.

†The strength is based on the nominal area of the rope. The figures given are only approximate and are based on 1-in rope sizes and $\frac{1}{8}$ -in aircraft-cable sizes.

Source: Compiled from *American Steel and Wire Company Handbook*.